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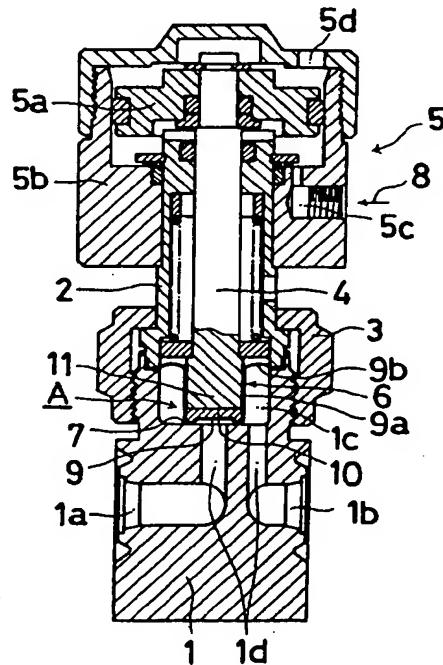
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⑯ Fluid flow controller.

⑯ A fluid flow control valve has the closure part of either the valve body (6) or the valve seat (7) faced by thin metallic film backed by a resiliently deformable body.

FIG. 1



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The invention relates to a fluid flow control assembly for use in semiconductor manufacturing equipment, vacuum equipment, medicine and food manufacturing equipment and the like, and more particularly, to a valve having good durability and sealing.

The closure parts of existing valves have generally involved metal to resin contact or metal to metal contact.

Metal to resin contact being superior in sealing performance and durability, has been extensively used for semiconductor manufacturing equipment, medicine and food manufacturing equipment and the like. However, the combination of metal and resin in valves involves a number of problems.

The first problem is the problem of gas substitution performance. Generally, once gas has entered into a narrow gap, it is extremely difficult to remove. In a closure having the combination of metal and resin, it is necessary to fix the resin to metal (for example, to fix the resin to the tip of a stem), causing a gap. Removal of the gas that has entered into the gap is difficult, prejudicing the performance of the valve in respect to gas substitution.

The second problem is the problem of discharging gas from the resin. Various gas components are contained in the resin and such components are desorbed with the rise in vacuum. As a result of this, reduced degree of vacuum or prolonged vacuuming time is caused by discharging the foregoing gas.

As a means for solving these problems it has been proposed to reduce the volume or the surface area of resin and to use improved materials, but these proposals have not been entirely successful.

A metal to metal closure does not present the foregoing problems. Therefore, arrangements such as shown in Figure 9 through Figure 11 have been used.

However, in the closure part A of the conical type as shown in Figure 9, because the valve body B slides against the valve seat C, pulverized particles are generated by sliding. In the flat type closure part A shown in Figure 10, the sealing performance between the valve body B and the valve seat C is relatively low. Further, in the protruding type closure part A shown in Figure 11, crushing, denting or flaking is likely to occur in the part where the valve body B and the valve seat C come in contact.

In a closure part having the combination of metals, in order to increase the sealing performance thereof, machining accuracy of the sheet surface can be increased, soft metal can be used, and the surface pressure of the sheet section should be increased among other conditions. Various types of restrictions are involved including the

rise in machining costs, generation of dust of pulverized particles and occurrence of crushing and deformation.

The present invention sets out to solve problems such as the foregoing.

According to the present invention there is provided a fluid flow control assembly with a closure part comprising a valve body and a valve seat which contact one another to shut off the fluid flow characterised in that the contact surface of either the valve body or the valve seat comprises a metallic film backed by a resiliently deformable body.

In the fluid flow control valve according to the invention when the valve is to be closed, the valve body moves to contact the valve seat. As closure force is applied the film is deformed to the configuration of the surface of which it contacts, this deformation being allowed by deformation of the resiliently deformable body.

As the closure force applied to the valve body is released, the film and the body having been subjected to slight elastic deformation restore their original shape.

The invention will now be described by way of example and with reference to the drawings wherein:-

Figure 1 is a longitudinal section of a fluid controller according to the first embodiment of the present invention.

Figure 2 is a section illustrating the closure part A of a fluid controller according to the second embodiment of the present invention.

Figure 3 is a section illustrating the closure part A of a fluid controller according to the third embodiment of the present invention.

Figure 4 is a section illustrating the closure part A of a fluid controller according to the fourth embodiment of the present invention.

Figure 5 is a section illustrating the closure part A of a fluid controller according to the fifth embodiment of the present invention.

Figure 6 is a section illustrating the closure part A of a fluid controller according to the sixth embodiment of the present invention.

Figure 7 is a section illustrating the closure part A of a fluid controller according to the seventh embodiment of the present invention.

Figure 8 is a section illustrating the closure part A of a fluid controller according to the eighth embodiment of the present invention.

Figures 9 to 11 are explanatory drawings of the closure parts A according to the prior art.

In the drawings A is the closure part, 1 the valve part with the fluid flow passage, 2 the cover, 3 the cover nut, 4 the valve stem, 5 the driving unit, 6 the movable valve body, 7 the valve seat, 8 hydraulic fluid for activation, 9 thin metallic film, 10

the resiliently deformable cushion body, 11 the valve body part which carries the film, and 12 the valve part which defines the seat.

Part 1 has fluid inlet 1a, fluid outlet 1b, a recess 1c, and fluid flow passages. Valve seat 7 is formed at the base of recess 1c. Cover 2 is securely installed on part 1 by tightening the nut 3. Further, inside the cover 2 is inserted the slideable stem 4 with valve body at the lower end thereof. A driving piston 5a is fixed to the upper end of stem 4 and drive unit 5 moves the piston unit 5.

The drive unit 5 comprises the piston 5a and cylinder 5b, and the stem 4 is vertically moved by pressure fluid 8 from the fluid supply inlet 5c or the fluid supply inlet 5d.

The closure part A which is a main feature of the present invention is comprised of valve body 6 and valve seat 7. Valve body 6 has an end portion 11 which is defined by the lower end of the stem 4. A thin metal film 9 covers base 11 and a resiliently flexible cushion body 10 is positioned between the valve body base 11 and film 9.

The thin metal film 9 is formed into a body having a cylindrical hollow section 9a to receive the valve stem and a rib section 9b. The film body 9 may be formed, for example by pressing a film of nickel or nickel alloy or by electrodeposition.

The circumferential edge of the rib section 9b is clamped between the part 1 and cover 2. The section 9a is installed in the recess 1c with the lower end of the stem 4 inserted. By tightening the cover 2 the airtightness of the valve recess 1c is maintained.

The cushion body 10 may be formed by a material or part which maintains appropriate elasticity and flexibility such for example as rubber, synthetic resin or a coil spring, in Figure 1 the body 10 is of fluoro-rubber formed into a disc.

Although in the embodiment of the present invention the cushion body 10 is preformed it can be formed in situ by pouring liquid resin into the space between 11 and the film.

As described above closure part A is incorporated into a piston driving mechanism. The fluid controller to which the closure part A is applied can be incorporated into other types of valves such as relief valves, check valves and pressure reducing valves.

Figure 2 shows the closure part A in a second embodiment of the present invention. The valve body 6 is formed by the valve body part 11 at the lower end of the stem 4. The thin metal film 9 is dished with a rim and the cushion body 10 is a coil spring. The cross section for inserting the body 10 comprises a groove 11a or part 11 and groove 9c on the body. Further, the rim of film 9 is welded to the external surface of the part 11.

In the embodiment of Figure 3 the body 10

again is inserted between the flat front surface of 11 and guide groove 9c in film 9.

In the embodiment of Figure 4 the dished film 9 has a contacting hemispherical rib 9d and the circumferential rim is welded to the external surface of the valve stem 4. Rubber or the like elastomer fills the space between the part 11 and film 9 to provide the cushion body.

In Figure 5 the lower part 11 of the valve body has fluid passage 11a at the centre in contrast to previous embodiments where the fluid passage is in valve seat 7. The film 9 is an annulus of U-shaped cross section and the circumferential edge is welded on the external surface of the valve body base 11, an annulus surrounding passage 11c.

In the embodiments of Figures 6 to 8 closure sections are shown in which the film 9 and body 10 are on the valve seat surrounding fluid flow passage 1d.

Thus in Figure 6 an annular cushion body 10 made of coil spring provides a shoulder around the valve seat and the ring-like cover film 9 made of thin metallic film covers the cushion body and is welded to the valve seat base 12.

According to Figure 7 the valve seat base 12 and the film 9 have co-operating hemispherical grooves 12a and 9c and ring-like cushion body 10 made of coil spring is inserted into these grooves.

According to Figure 8 a filler such as rubber occupies the space between the valve seat base 12 and the film 9. The contacting section 11b of the valve has an almost semispherical cross section to contact the film 9.

In the foregoing embodiments, the circumference of the end of the film 9 is welded to the valve body base 11 or the valve seat base 12, but the film 9 can be tightly installed by the method other than welding such for example as adhesion or pressure welding and caulking.

The valve body base 11, valve seat base 12, the body of film 9 and resiliently flexible cushion body 10 may take shapes other than that as shown in the foregoing embodiments as long as the film 9 is contacted elastically to the valve seat base 12 or the valve body base 11 by means of the cushion body 10.

The operation of the fluid controller according to the present invention will now be explained in regard to the embodiments having the resilient contacting surface of body 10 and film 9 on the moving valve member.

The stem 4 is activated by the driving unit 5, and when the valve body 6 is lowered, the covered body of film 9 contacts the valve seat 7. Then, as the stem 4 continues to lower by a small distance, the cover body 9 is pushed against the top of the valve seat movement being allowed by resilient yielding or elasticity, the external surface of the

film 9 seats in a slightly deformed condition along the profile of the external surface of the valve seat 7. This ensures a tight closure.

At that time, because the cover body 9 is formed by the thin film of metal, the cover body 9 closely contacts the external surface of the valve seat 7, and high adhesion with good conformability can be obtained.

In addition, the cover body 9 contacts elastically with the valve seat 7 by means of the cushion body 10, and even if a relatively large surface pressure is applied, the external surface of the valve seat 7 and the cover body 9 is free from defects such as dents and generation of the dust of pulverized particles caused by friction.

By pulling up the valve body 6, the cover body 9 and the cushion body 10 therein which have been slightly elastically deformed by the surface pressure applied are restored to their initial shape. By pulling up the valve body 6 continuously, the cover body 9 is separated from the valve seat base 12, allowing fluid passage.

The following further advantages are achieved:-
(1) Because the cover body made of thin metallic film contacts gently on the valve seat surface (or on the valve body) with elasticity, the valve seat surface and other surfaces are hardly scratched, the film is not damaged and its life is therefore extended, little dust of pulverized particles is caused by friction thus improving the cleanliness considerably.

(2) Even if the cushion body is made of rubber and the like, the discharging of the internal gas is completely shut off by the cover body made of thin metallic film, and even if the cover body is applied to a vacuum device, the reduction of the degree of vacuum caused by discharging of gas never will not occur.

(3) Because the valve body base 11 or the valve seat base 12 is completely covered by the cover body made of thin metallic film, the combined space such as the one in the sheet section A related to the conventional combination of resin and metal is eliminated, what is called the gas substitution performance is remarkably improved, and the material of the cushion body can be selected without regard to the type of the fluid.

(4) The metallic film can be relatively easily processed with surface treatment or forming, and thus the sheet A of high accuracy can be formed at a low price. Such sheet A can easily stand high temperatures.

(5) Because the shape of the cover body is relatively simple, the material of thin metallic film to be used is subjected to almost no restrictions in view of workability, and the material of thin metallic film can be selected appropriately

according to the type of the fluid.

(6) Because high sealing performance can be obtained at relatively low surface pressure, considerable reduction of size of the driving system of the fluid controller becomes possible.

Claims

1. A fluid flow control assembly with a closure part comprising a valve body 6 and a valve seat 7 which contact one another to shut off the fluid flow characterised in that the contact surface of either the valve body (6) or the valve (7) seat comprises a metallic film 9 backed by a resiliently deformable body 10.
2. An assembly according to Claim 1 wherein the contact surface is provided by an annular rib in the metallic film and the resiliently deformable body is inserted in the annular space defined between the rib and the valve body or seat.
3. A valve according to Claim 2 wherein the valve body or seat has an annular recess to cooperate with the rib to define the space for the body.
4. A valve according to Claim 3 wherein the resiliently deformable body is a coil spring.
5. A valve according to Claim 2 wherein the resiliently deformable body also extends behind the non-protruding part of the film.
6. A valve according to Claim 5 wherein the resiliently deformable body is a cushion of rubber or similarly deformable material.
7. A valve according to Claim 1 wherein the valve body comprises an annular rib and the valve seat comprises a shoulder around the fluid flow passage such shoulder being formed of the metallic film backed by the resiliently deformable body.
8. A valve according to any of the preceding claims wherein the metallic film is formed of nickel or nickel alloy.
9. A valve according to Claim 1 wherein the metal film extends around the valve body to be sealed to a fixed part of the valve assembly.
10. A valve according to Claim 1 wherein the metal film is discharged having a rim fixed to the valve body.

FIG. 1

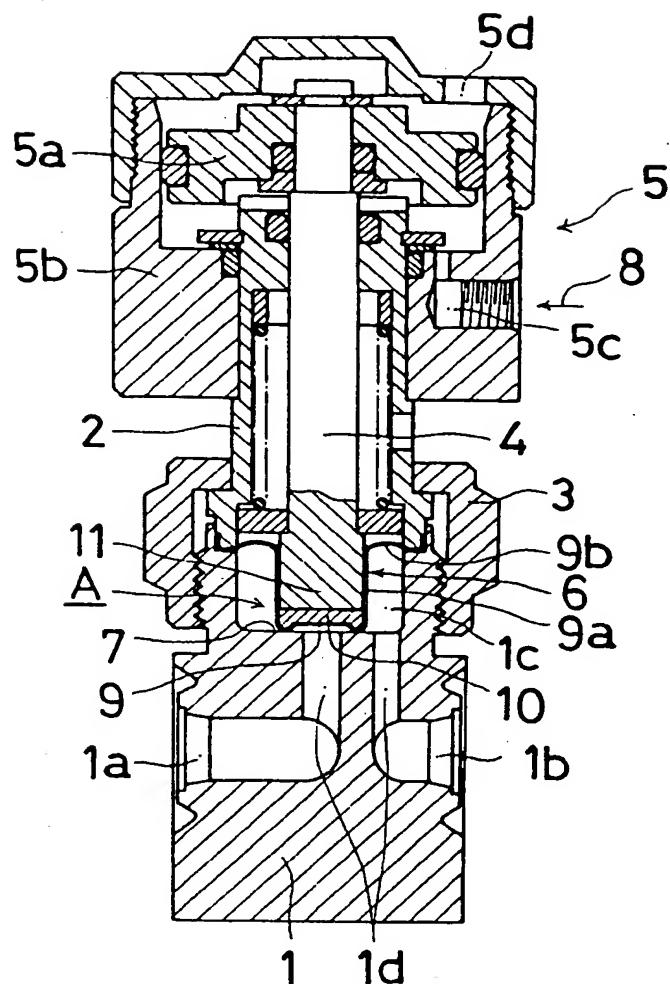


FIG. 2

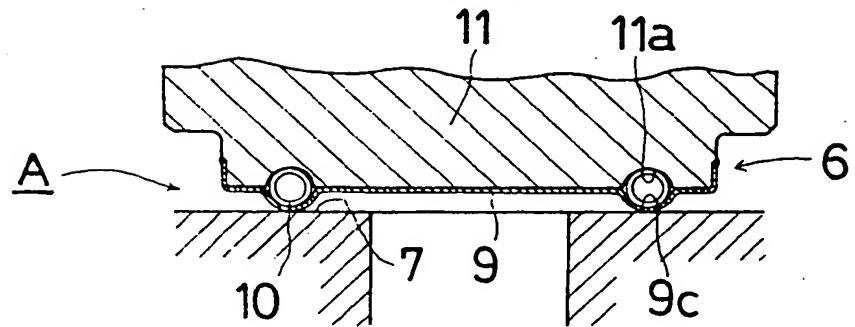


FIG. 3

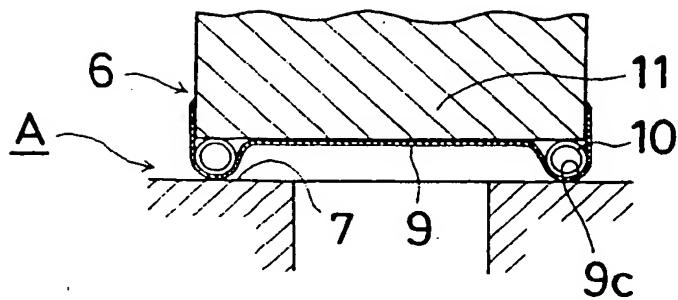


FIG. 4

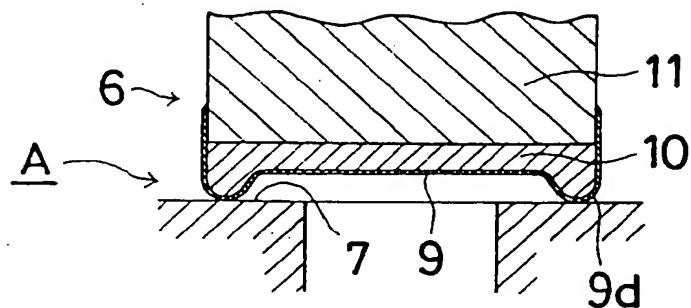


FIG. 5

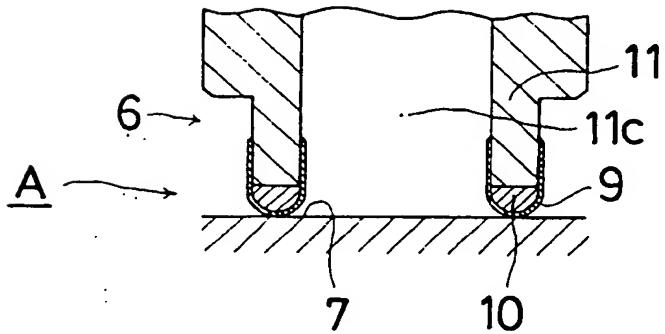


FIG. 6

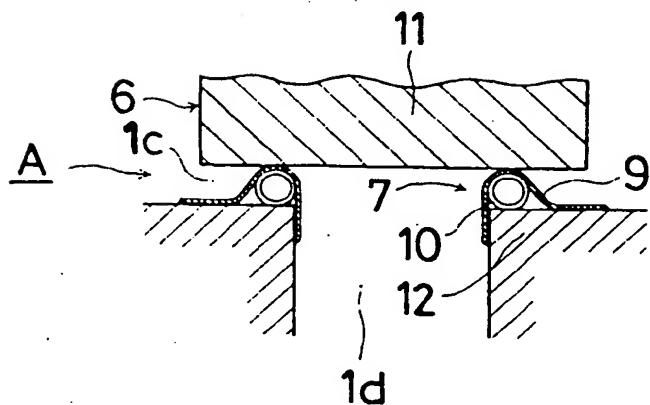


FIG. 9

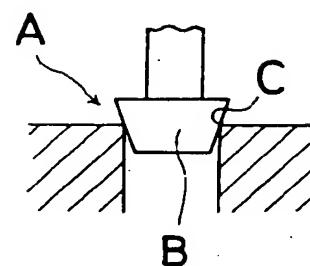


FIG. 7

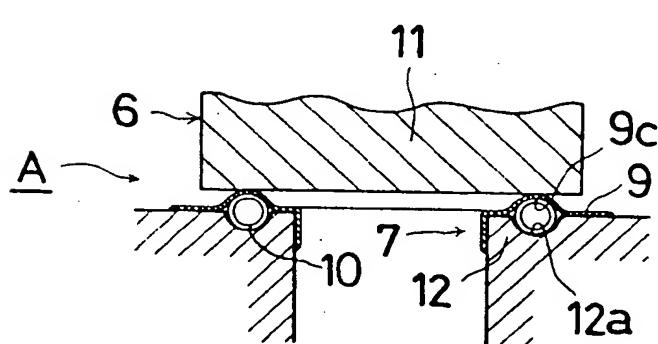


FIG. 10

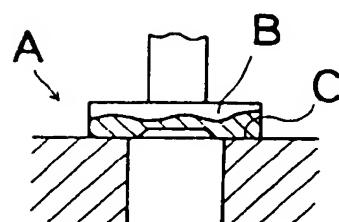


FIG. 8

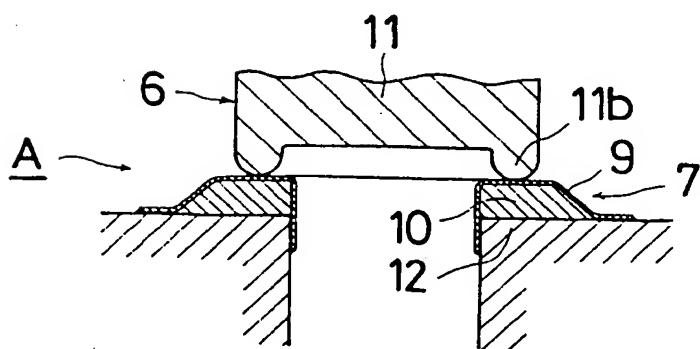
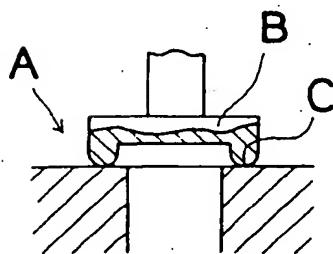
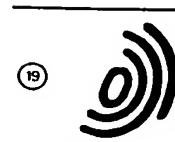


FIG. 11





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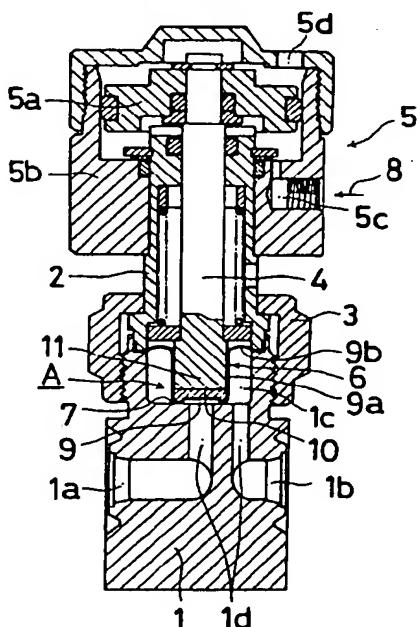
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⑯ Fluid flow controller.

⑯ A fluid flow control valve has the closure part of either the valve body (6) or the valve seat (7) faced by thin metallic film backed by a resiliently deformable body.

FIG. 1



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EUROPEAN SEARCH REPORT

Application Number

EP 92 30 0450

DOCUMENTS CONSIDERED TO BE RELEVANT			CLASSIFICATION OF THE APPLICATION (Int. Cl.5)
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	
X	DE-A-2 724 790 (REALMECA) * claims 12,21,25; figures 5,7 *	1-7,10	F16K 1/34 F16K 25/00
A	DE-A-2 454 452 (VALVE SYSTEMS INTERNATIONAL) * page 20, line 8 - line 15; figure 17 *	1-4,10	
A	DE-C-659 838 (AEG) * claim 1; figures 1-4 *	1,2,5,6	
A	DE-C-690 379 (SCHAFFER & BUDENBERG) * page 2, line 55 - line 58; claims 1,2; figure 1 *	1,3,6,8	
A	DE-A-2 038 905 (HILLS-MCCANNA) * page 18, line 25 - page 21, line 17; figures 11-13 *	1,2,7,10	
A	EP-A-0 069 034 (CEFILAC) * page 6, line 11 - line 14 * * page 9, line 34 - line 35 *	1	
A	US-A-3 508 739 (ALLGEIER) * column 3, line 25 - line 27; figure 4 *	6	TECHNICAL FIELDS SEARCHED (Int. Cl.5)
A	US-A-3 399 695 (STEHLIN) * figures 1,2 *	9	F16K
The present search report has been drawn up for all claims			
Place of search	Date of compilation of the search	Examiner	
BERLIN	30 JULY 1992	SCHLABBACH	
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